

# TSRT09 – Control Theory

Lecture 4: Kalman filter &  
the closed-loop system

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## Summary of lecture 3

- Covariance matrix (at  $\tau$ ):

$$R_u(\tau) = E[u(t)u(t - \tau)^T] = \int_{-\infty}^{+\infty} u(t)u(t - \tau)^T d\tau$$

- Spectrum:

$$\Phi_u(\omega) = \int_{-\infty}^{\infty} R_u(\tau)e^{-i\omega\tau} d\tau$$

- Signal “size” (covariance matrix at 0):

$$R_u = R_u(0) = \frac{1}{2\pi} \int_{-\infty}^{\infty} \Phi_u(\omega) d\omega$$

## Summary of lecture 3 (cont'd)

- White noise  $v(t)$ :

$$\Phi_v(\omega) = \text{constant} = R$$

$$R_v(\tau) = R \delta(\tau)$$

- Spectra factorization: every spectrum can be thought as being generated by a white noise passed through a linear system

$$y = Gv \quad \Rightarrow \quad \Phi_y(\omega) = G(i\omega) \underbrace{\Phi_v(\omega)}_R G^T(-i\omega)$$

## Summary of lecture 3 (cont'd)

- White noise in state space form:

$$\dot{x} = Ax + Nv$$

$v$  white noise of intensity/spectrum  $R$ .

- Covariance matrix of  $x$ :  $\Pi_x = R_x(0)$  solution of

$$A\Pi_x + \Pi_x A^T + NRN^T = 0 \quad \text{Lyapunov equation}$$

## Summary of lecture 3 (cont'd)

- System

$$\begin{aligned}\dot{x} &= Ax + Bu + Nv_1, \\ y &= Cx + Du + v_2\end{aligned}$$

$v_1, v_2$  white noises

- Observer

$$\dot{\hat{x}} = A\hat{x} + Bu + K(y - C\hat{x} - Du)$$

- Estimation error

$$\dot{\tilde{x}} = (A - KC)\tilde{x} + Nv_1 - Kv_2$$

## Lecture 4

- Kalman filter

In the book: Ch. 5

- PART II: LINEAR CONTROL THEORY
  - Description of the closed-loop system

In the book: Ch. 6

# Kalman filter

- System

$$\dot{x} = Ax + Bu + Nv_1,$$

$$y = Cx + Du + v_2$$

$v_1, v_2$  uncorrelated white noises with intensity  $R_1, R_2$

- Observer:  $\dot{\hat{x}} = A\hat{x} + Bu + K(y - C\hat{x} - Du)$

**Kalman filter** = optimal observer, of gain:

$$K = PC^T R_2^{-1}$$

where  $P$  is given by the **algebraic Riccati equation** (ARE):

$$AP + PA^T - PC^T R_2^{-1} CP + NR_1 N^T = 0$$

- It minimizes covariance of estimation error  $P = E[\tilde{x}(t)\tilde{x}^T(t)]$  subject to  $A - KC$  stable

# When can the algebraic Riccati equation be solved?

If

1.  $R_1 \geq 0, R_2 > 0$
2.  $(A, C)$  detectable

Meaning:

- the unstable part of the system is observable
- observer error (i.e.,  $A - KC$ ) can be rendered stable

3.  $(A, NR_1N^T)$  stabilizable

Meaning:

- the unstable part of the system is "controllable from noise"
- more mathematical/technical condition

then there is a solution  $P = P^T \geq 0$  to the algebraic Riccati equation  
 s. t. all eigenvalues of  $A - KC = A - PC^T R_2^{-1} C$  have real part  $< 0$ .



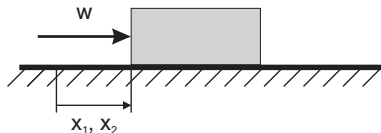
# Properties of the Kalman filter

1. Kalman filter is the observer that minimizes the covariance matrix  $P$  for the estimation error  $\tilde{x}$
2. Kalman filter minimizes the mean square error (i.e., the variance of  $\tilde{x}$ ) among all causal filters
3. Error  $\tilde{x}(t)$  is uncorrelated with estimates  $\hat{x}(s)$ ,  $s \leq t$
4. "Innovation":  $\nu(t) = y(t) - C\hat{x}(t) - Du(t)$  is a white noise (completely unpredictable)

# Application of Kalman filter: sensor fusion

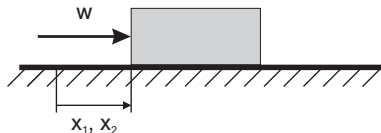
- Often one has to consider the same signal from different sensors having different accuracies.
- In a navigation system for ship, airplane, car, .... one considers simultaneously
  - Position information: GPS, radio beacons, optical signals...
  - Velocity information: doppler radars, tachometers, gyro (angular velocity),...
  - Acceleration measurements: accelerometers,....
  - .....
- An important point in this context is that the weak aspects of a sensor can be compensated by another sensor that does not have the *same* weak aspects.

## Simple sensor fusion: position – velocity



- Movement in 1-dimension:  $x_1$  position,  $x_2$  velocity.
- Acceleration varies randomly around 0: modeled as a white noise,  $w$ .
- Position and velocity measurement:  $y_1$  resp.  $y_2$ .
- Position and velocity are measured with some measurement error:  $v_1$  resp.  $v_2$ .

# Simple sensor fusion: position – velocity



Model:

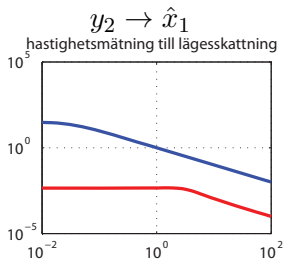
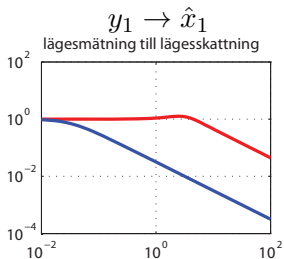
$$\dot{x} = \begin{bmatrix} 0 & 1 \\ 0 & 0 \end{bmatrix} x + \begin{bmatrix} 0 \\ 1 \end{bmatrix} w$$

$$y = x + v$$

$$R_1 = 1, \quad R_2 = \begin{bmatrix} r_1 & 0 \\ 0 & r_2 \end{bmatrix}$$

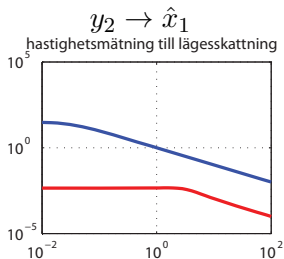
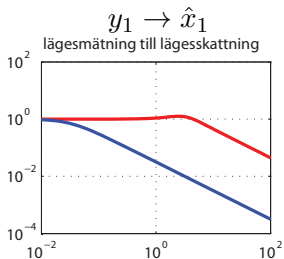
$w, v$  uncorrelated

# Kalman filter: Bode diagram - Position estimation



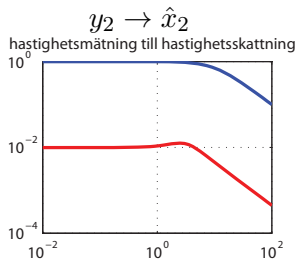
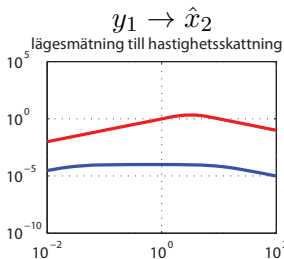
- Good position meas. & bad velocity meas.  
( $r_1$  small,  $r_2$  big):  $\hat{x}_1 \approx y_1 + 0 \cdot y_2$
- Bad position meas. & good velocity meas.  
( $r_1$  big,  $r_2$  small):  $\hat{x}_1 \approx 0 \cdot y_1 + \int y_2$

# Kalman filter: Bode diagram - Position estimation



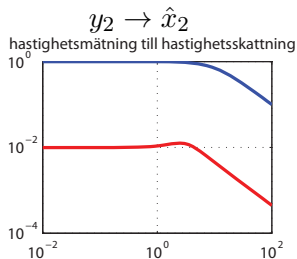
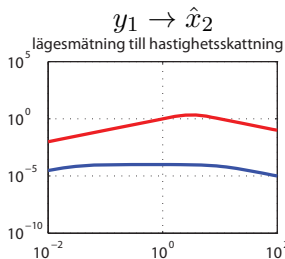
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# Kalman filter: Bode diagram - Velocity estimation



- Good position meas. & bad velocity meas.  
( $r_1$  small,  $r_2$  big):  $\hat{x}_2 \approx \frac{d}{dt}y_1 + 0 \cdot y_2$
- Bad position meas. & good velocity meas.  
( $r_1$  big,  $r_2$  small):  $\hat{x}_2 \approx 0 \cdot y_1 + y_2$

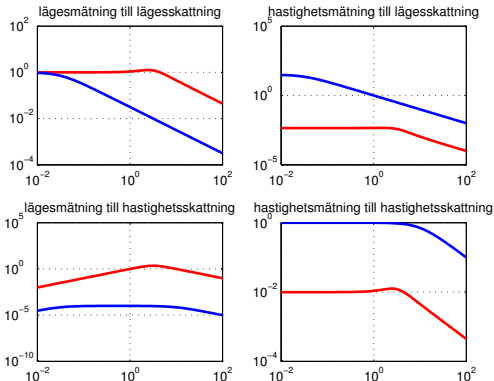
# Kalman filter: Bode diagram - Velocity estimation



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( $r_1$  small,  $r_2$  big):  $\hat{x}_2 \approx \frac{d}{dt} y_1 + 0 \cdot y_2$
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( $r_1$  big,  $r_2$  small):  $\hat{x}_2 \approx 0 \cdot y_1 + y_2$



# Kalman filter: Bode diagram - Summary



In other words: high noise level in a measurement  $\Rightarrow$  Filter does not "trust" too much that measurement

## PART II: LINEAR CONTROL THEORY

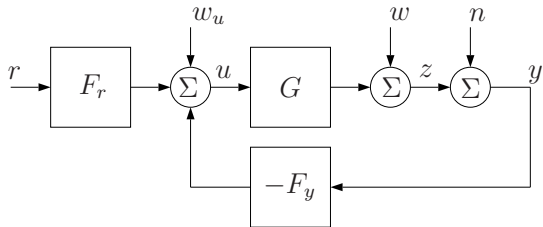
- Lecture 4: The closed-loop system
- Lecture 5: Controller structure
- Lecture 6: LQ-regulation
- Lecture 7: Loopshaping
- Lecture 12: Specifications and limitations
  - Canonical block diagram
  - Stability for the closed-loop system
  - Sensitivity & Robustness
  - Specifications

## PART II: LINEAR CONTROL THEORY

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# The closed-loop system: important signals



## Signals

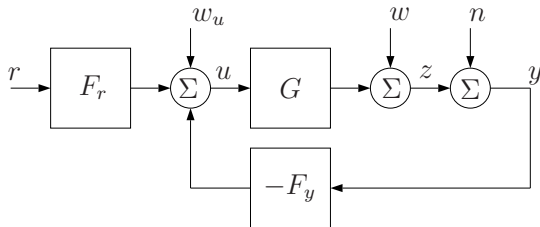
- $r$  = reference signal
- $u$  = control signal
- $z$  = regulated variable
- $y$  = measured signal

## Disturbances

- $w$  = system disturbance (at the output)
- $n$  = measurement noise
- $w_u$  = disturbance at the input

- Often it is  $y = z + n$
- **Control law:**  $u = F_r(s)r - F_y(s)y$

# The closed-loop system: important transfer functions



Relationships among signals:

$$z = G_c r + S w - T n + G S_u w_u$$

$$u = S_u F_r r - S_u F_y (w + n) + S_u w_u$$

# The closed-loop system: important transfer functions

Transfer functions:

- **Closed-loop system** (“main T.F.”)

$$G_c = (I + GF_y)^{-1}GF_r$$

- **Sensitivity function**

$$S = (I + GF_y)^{-1}$$

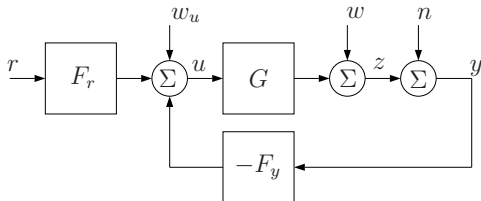
- **Complementary sensitivity function**

$$T = (I + GF_y)^{-1}GF_y$$

- **Input sensitivity function**

$$S_u = (I + F_yG)^{-1}$$

# Stability for the closed-loop system



- Basic control design: choose  $F_y$  so that  $G_c$  is stable
- Q: in presence of disturbances, is it enough to have  $G_c$  stable?
- Input-output stability: stability from all inputs ( $r, w, n, w_u$ ) to all signals ( $u, z, y$ )
- Q: when is the closed loop system stable from all inputs to all outputs?

## Stability for the closed-loop system

- Consider the closed-loop system as a system with inputs  $w_u, w$  and output  $u, y$  (for  $r = 0, n = 0$ ).

$$\begin{bmatrix} y \\ u \end{bmatrix} = \begin{bmatrix} SG & S \\ S_u & -S_u F_y \end{bmatrix} \begin{bmatrix} w_u \\ w \end{bmatrix} \quad (1)$$

- "Internal stability": if the 4 transfer functions

$$SG, S, S_u, S_u F_y$$

are all stable (and  $F_r$  stable) then the entire system is stable



## Sensitivity (to model errors)

- In practice the system is not known exactly.
- nominal system:  $G$
- true system:  $G_0 = (I + \Delta_G)G$      $\Delta_G =$  model error
- true output  $z_0 = (I + \Delta_z)z$
- relative output error

$$\Delta_z = S_0 \Delta_G, \quad S_0 = (I + G_0 F_y)^{-1}$$

- In practice: since  $G_0$  is not known,  $S_0$  must be approximated by

$$S_0 \simeq S = (I + G F_y)^{-1}$$

- Interpretation:  $S$  is the gain from model error to rel. output error  
 $S$  should be small when the model error  $\Delta_G$  is big

## Robustness (to model errors)

- How big a model error  $\Delta_G$  can be tolerated without compromising the stability of the closed-loop system?

- If

$$\|\Delta_G T\|_\infty < 1$$

then the closed-loop system is still stable.

- In its turn, this is true if

$$|T(i\omega)| < \frac{1}{|\Delta_G(i\omega)|}, \quad \text{all } \omega$$

- Interpretation:  $T$  should be small when the model error  $\Delta_G$  is big

## Specifications for control design

**Task:** Choose  $u$  so that  $z$  follows  $r$  as close as possible, in spite of the presence of disturbances  $w, n$ , and of uncertainty in the system, while at the same time using reasonable values of  $u$

Translate this into transfer functions:

- error  $e = r - z$  small

$$e = (I - G_c)r - Sw + Tn$$

- input not too large

$$u = G_{ru}r + G_{wu}(w + n)$$

- effect of uncertainty

$$\Delta_z = S_0\Delta_G \sim S\Delta_G$$

- closed-loop stability with model error

$$\|\Delta_G T\|_\infty < 1$$

# Specifications for control design

Conditions:

- $I - G_c$  small  $\Rightarrow$  controlled variable  $z$  follows reference signal  $r$
- $S$  small  $\Rightarrow$  system disturbance and model errors have small impact on the controlled variable
- $T$  small  $\Rightarrow$  measurement disturbance has small impact on the controlled variable, and model errors do not compromise stability
- $G_{ru}$  and  $G_{wu}$  small  $\Rightarrow$  input  $u$  stays moderate

But observe that

$$S + T = I$$

$$G_c = GG_{ru}$$

$\Rightarrow$  **conflicts on the specifications!**

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Lecture 4

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