

## Summary

The *static* friction phenomena in a manipulator joint is analyzed in detail regarding the influences caused by:

**Joint position**  $\Rightarrow$  related to asymmetries in the joint contact surfaces.

**Joint load torques**  $\Rightarrow$  related to the increase of adhesion between the contact surfaces in the joint.

**Temperature**  $\Rightarrow$  related to variations of lubricant viscosity and contact surfaces interaction.

A series of experiments was carried out to consider the effects of each variable independently. According to the empirical observations a *new static friction model*, based on a standard velocity dependent model, is proposed to achieve an overall improved performance.

## Friction Curve Estimation

Consider the manipulator rigid-body model:

$$M(\varphi_a) \ddot{\varphi}_a + C(\varphi_a, \dot{\varphi}_a) + \tau_g(\varphi_a) + \tau_f(\dot{\varphi}_m) = u$$

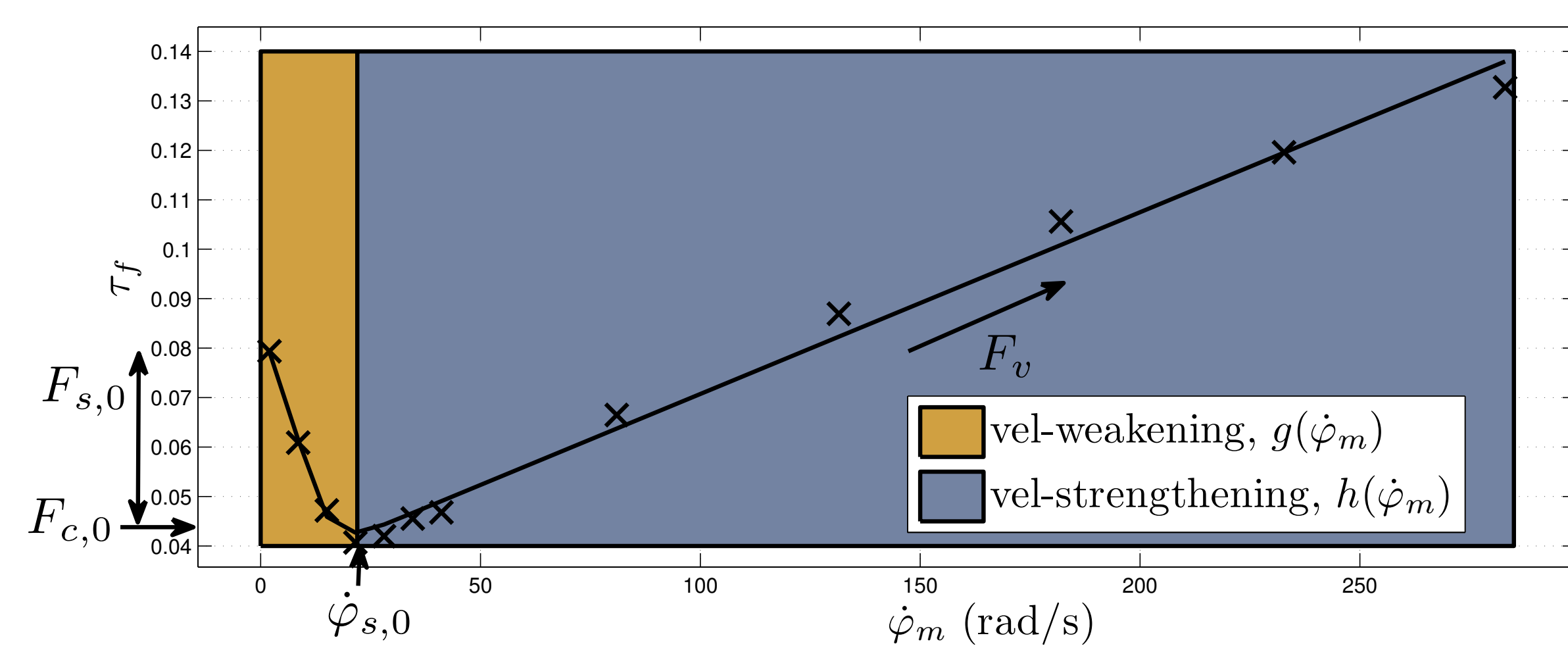
Moving *one axis at a time in steady-state velocity*, then

$$\ddot{\varphi}_a \approx 0, \quad C(\varphi_a, \dot{\varphi}_a) = 0$$

Assume  $\tau_f(-\dot{\varphi}_m) = -\tau_f(\dot{\varphi}_m)$  (direction independence). Take movements over the same position  $\bar{\varphi}_a$  in forward  $u^+$  and backward  $u^-$  directions, for a constant speed  $\bar{\varphi}_m$ :

$$\begin{aligned} \tau_f(\bar{\varphi}_m) + \tau_g(\bar{\varphi}_a) &= u^+, \\ \tau_f(-\bar{\varphi}_m) + \tau_g(\bar{\varphi}_a) &= u^- \end{aligned} \Rightarrow \tau_f(\bar{\varphi}_m) = \frac{u^+ - u^-}{2} \quad (1)$$

Moving the joint through several steady-state velocities back and forth, it is possible to estimate a static *friction curve* using (1).

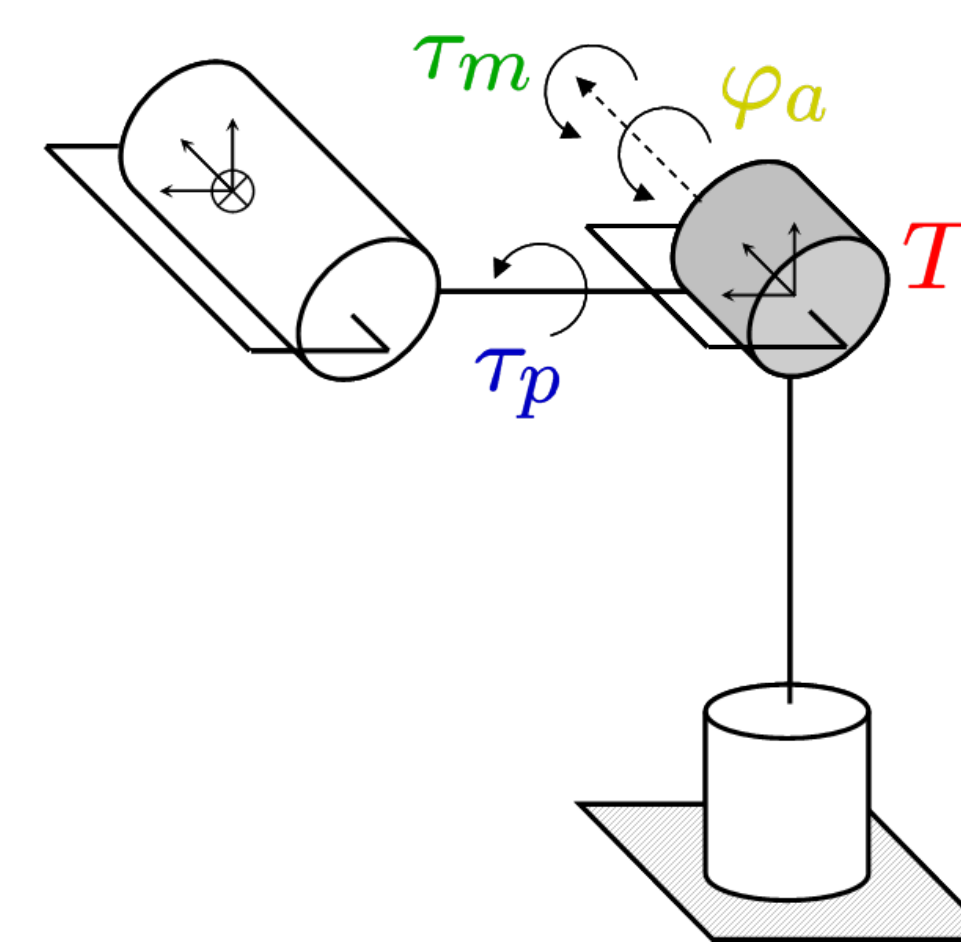


Estimated static friction curve and  $\mathcal{M}_0$  model-based predictions.

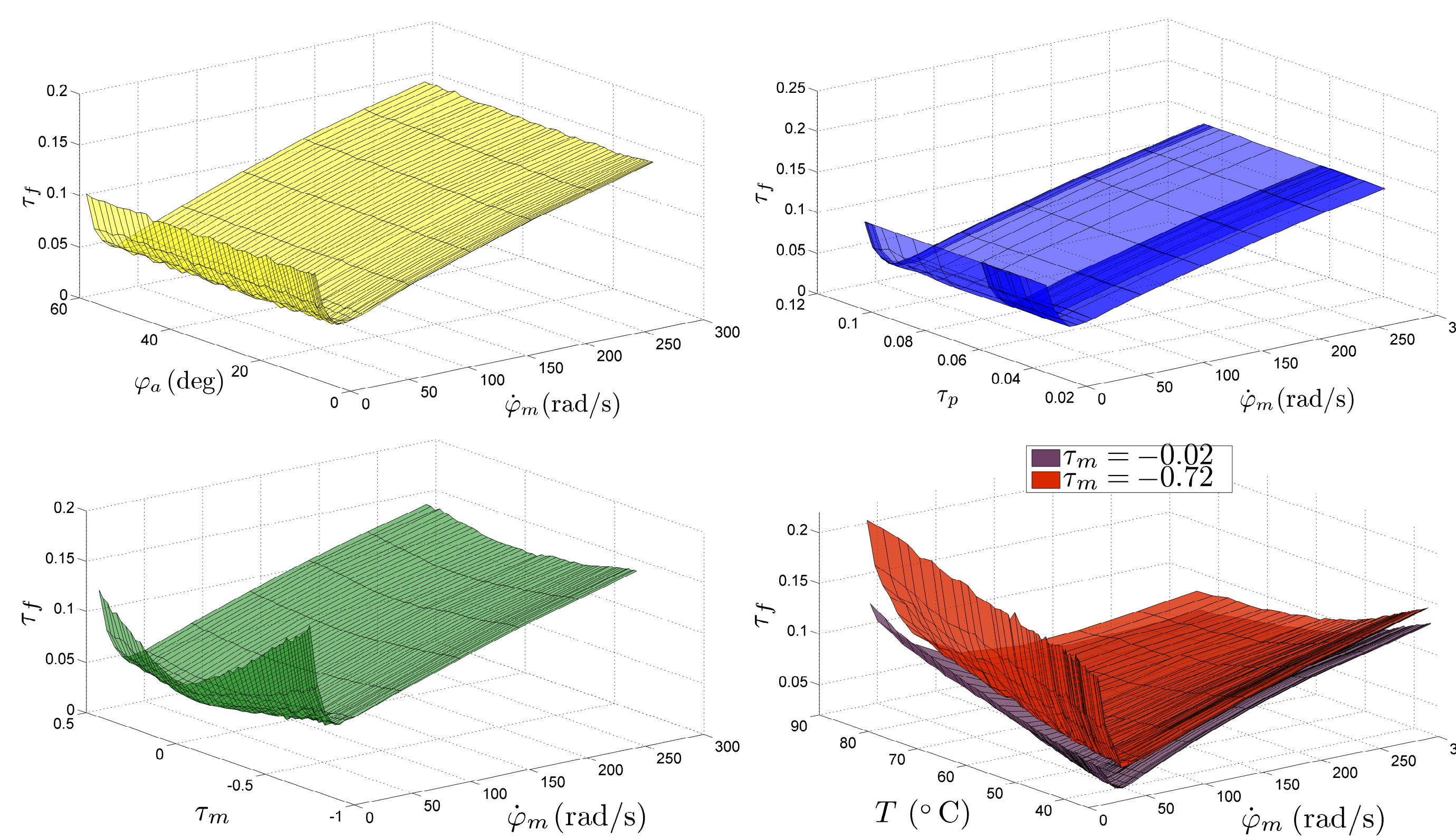
## Influences Analysis

Dedicated experiments at one of the joints of a 6 axes industrial robot were taken in order to analyze the joint friction influences caused by

- Position**  $\varphi_a$ : arm positions.
- Perp. load**  $\tau_p$ : resulting perpendicular load torque.
- Manip. load**  $\tau_m$ : resulting manipulated load torque.
- Temperature**  $T$ : joint temps.



Special care was taken to *avoid combined effects* during the experiments. Using a model, a set of configurations is chosen a priori in order to control variations of  $\varphi_a$ ,  $\tau_p$  and  $\tau_m$ . Changes in  $T$  are achieved with a warm up cycle and cooling down.



## Main Effects

- $\varphi_a$ : no sensible effects.
- $\tau_p$ : no sensible effects (only small variations possible).
- $\tau_m$ : considerable *increase in the vel-weakening* regime.
- $T$ : considerable *increase in the vel-weakening* regime and considerable *decrease in the vel-strengthening*.

## Empirical Modeling

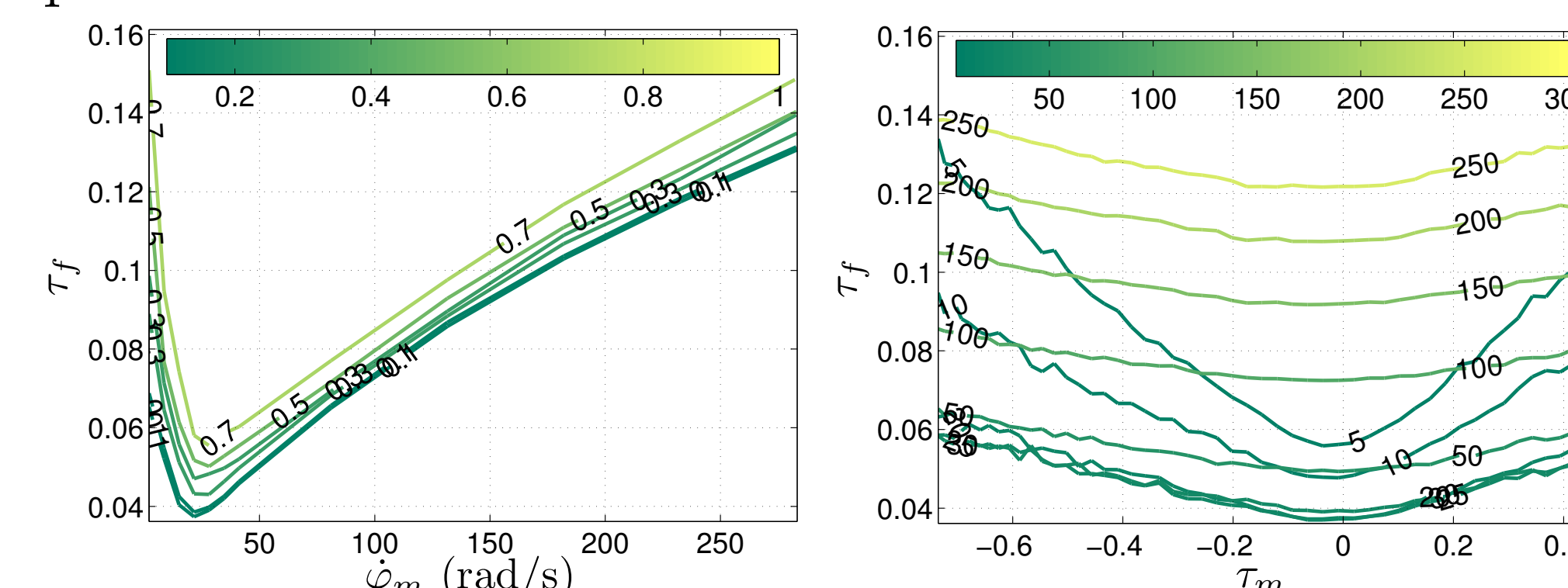
A standard parametrization of the static friction curve w.r.t.  $\dot{\varphi}_m$  is

$$\tau_f(\dot{\varphi}_m) = g(\dot{\varphi}_m) + h(\dot{\varphi}_m) = [F_c + F_s e^{-|\frac{\dot{\varphi}_m}{\dot{\varphi}_{s,0}}|^{1.3}}] + [F_v \dot{\varphi}_m] \quad (\mathcal{M}_0)$$

The effects of  $\tau_m$  and  $T$  are analyzed further to argue about an extension of  $\mathcal{M}_0$  that can cope with them.

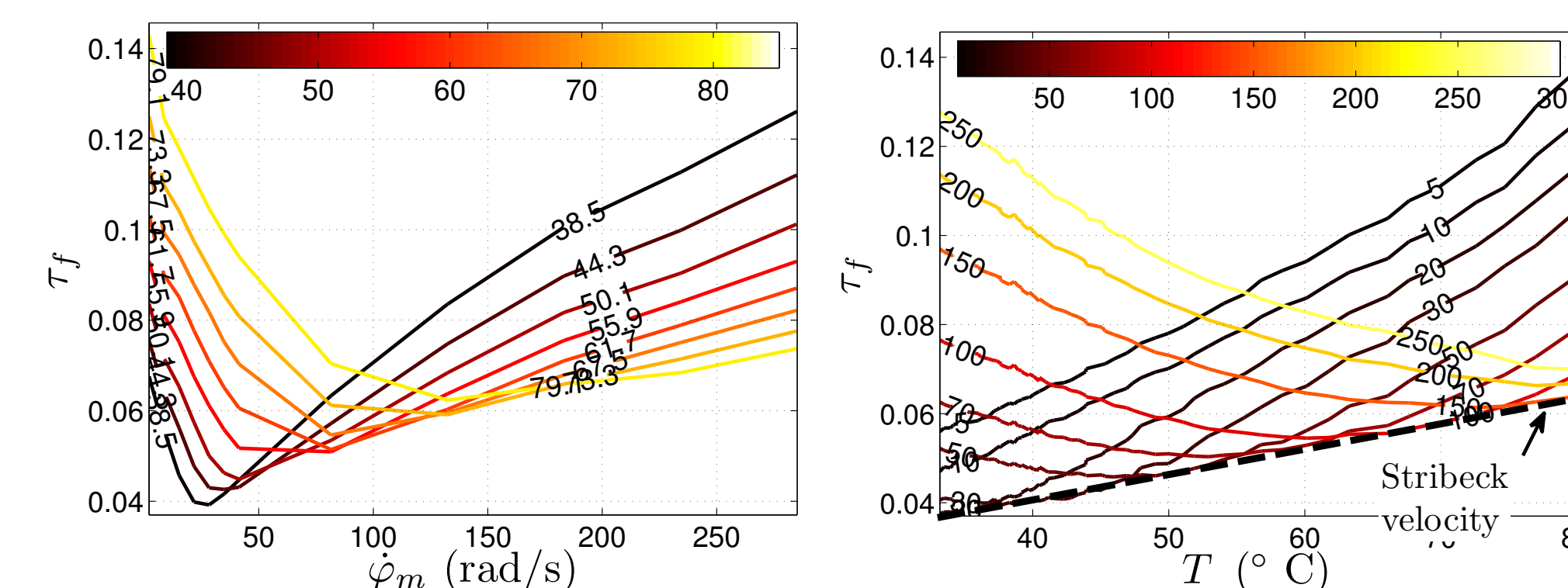
Effects of  $\tau_m$

- $F_c$  linear inc
- $F_s$  linear inc



Effects of  $T$

- $F_s$  linear inc
- $\dot{\varphi}_s$  linear inc
- $F_v$  exp. dec



Based on the observed behavior, the extended model  $\mathcal{M}^*$  is proposed

$$\begin{aligned} \tau_f(\dot{\varphi}_m, \tau_m, T) &= \left[ \{F_{c,0} + F_{c,\tau_m} |\tau_m|\} + F_{s,\tau_m} |\tau_m| e^{-|\frac{\dot{\varphi}_m}{\dot{\varphi}_{s,\tau_m}}|^{1.3}} \right. \\ &\quad \left. + \{F_{s,0} + F_{s,T} T\} e^{-|\frac{\dot{\varphi}_m}{\dot{\varphi}_{s,0} + \dot{\varphi}_{s,T} T}|^{1.3}} \right] + \left[ \{F_{v,0} + F_{v,T} e^{-\frac{T}{T_0}}\} \dot{\varphi}_m \right] \quad (\mathcal{M}^*) \end{aligned}$$

The parameters of  $\mathcal{M}^*$  are identified and it is validated over a wide operating range, reducing the average error *a factor of 6* when compared to  $\mathcal{M}_0$ .

